

# Objective and Solution

To perform the distributed launch mission structure for Artemis and future deep-space missions, it is critical to produce a reliable, low-loss, and low-complexity propellant storage and transfer system optimized for microgravity. While current propellant tanks primarily utilize passive liquid acquisition devices and helium pressurization methods, environmental differences and evolving needs have created a desire to explore new alternatives.

CROSS aims to reinvent the propellant storage tank by combining old ideas with a unique innovation.

Controlled boil-off of cryogenic fuel provides pressure stabilization without the need for helium pressurant Piston within storage tank provides adjustable volume control for slosh reduction and prevents fuel contamination Autogenous pressurant ramping provides ample force to propel the piston for fuel transfer



## Propellant Ullage-Driven Liquid Storage and Expulsion (PULSE) Test Bench:

### Pressure Differential Control Scheme

Requires multiple pressure readings for comparison between tank compartments ■Gradual increase in pressure for LN<sub>2</sub> expulsion within adequate time frame

### Pneumatic-Based Piston System

- Minimizes mechanical complexity
- Separates states of matter
- Reduced maintenance and risk of failure

### PULSE Testing Goals

Validate the CROSS design by demonstrating the viability of combining positive expulsion systems with autogenous pressurization methods Determination of experimental siphon ratio to better calculate theoretical CROSS expulsion efficiency

# Cryogenic Orbital Siphoning System CROSS Concept of Operations Stage 2: Stage 3: Coupling

CROSS pre-pressurized before installation to payload vehicle

Stage 1: Storage

- Valve minimally opened for limited flow of propellant to CVAPS
- CROSS stays pressurized over time
- Signals from the destination vessel are sent to the instrument system
- Automated couplers activate between destination vessel and CROSS

# CROSS Design

The Cryogenic Orbital Siphoning System utilizes a thermodynamic cycle to control boil-off within the storage tank, redirecting it to the other side of a piston to be used as a positive expulsion device. The key features include: **Expulsion Mechanism** 

- Pneumatic Pressure Control Piston
- Pressurization through Thermodynamic Cycling **CVAPS** Subsystem
- Heated Gradient Lines
- Heating Element For Vapor Saturation
- Single-Phase Compressor
- Expansion Tank For Pressure Storage
- Phase Separation for Liquid Propellant Transfer
- Siphoning Loop to Isolate Phases
- Centrifugal Cyclonic Separator



- Storage
- Reduces Manual
- Intervention and Error
- Stage 1: Isobaric Siphoning Stage 3: **Preheating to To Destination Single Phase** Stage 2: Vesse Vapor (Isenthalpic Expansion Stage 4: Vapor Stage 0: Compression Cryogenic Stage 5: Storage **Transfer to Expansion Tank** Stage 6: Stored Stage 7: Expansion Piston Readiness Actuation



gas

- Valve opens further to allow increased flow to CVAPS
- Increased rate of vaporization heightens gas pressure
- Piston begins to move in the direction of the exit point
- Automation and Feedback Pressure Sensing in Main
  - Automated Valve Control
- for System Response Maintains Boil-off During



Pressure equalizes at the

end of piston stroke

CROSS is now filled with

Destination vessel supply

is now fully replenished

CROSS disengages from

destination vessel

Schematic Of A Cyclonic Separator<sup>1</sup>

Pressure-Volume Relationships	
Stage 0	P <sub>o</sub> , V <sub>o</sub>
Stage 1	P <sub>1</sub> < P <sub>0</sub> , V <sub>1</sub>
Stage 2	$P_2 = P_1, V_1$ $\Delta h = 0$ (throttling)
Stage 3	P <sub>3</sub> > P <sub>4</sub> , V <sub>3</sub>
Stage 4	P <sub>4</sub> V <sub>4</sub> <sup>γ</sup> = Constant γ ~ 1.42
Stage 5	$P_{5} = P_{4}, V_{5}$
Stage 6	P <sub>6</sub> < P <sub>5</sub> , V <sub>6</sub> = V <sub>5</sub>
Stage 7	P <sub>7</sub> < P <sub>6</sub> , V <sub>7</sub>

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- CDR Development Marg
- **CROSS**

Mechanism Costs

Additional Costs

## Acknowledgements and References

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# **CROSS** Simulation

### **Apparatus Verification**

e preliminary pressure differential readings

- pressure loss effects on transfer speed
- ize transfer line fluid behavior

e simulations from ranging flow velocities

### ehavior

ence modeled within destination tank

- ent dissipation studied from flow simulation
- severely complex nature in simulation requiring parameter refinement





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